

Fig. 1 Comparison between the attenuation predicted by the general attenuation law developed in the course of the present study and the experimental results of Sommerfeld.³

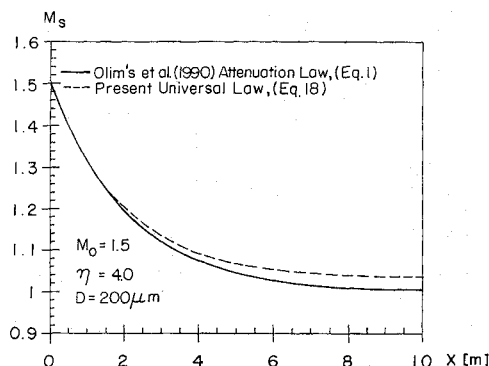


Fig. 2 Comparison between the predictions of the attenuation laws developed during the course of the present study (dashed line) and that developed by Olim et al.¹

as mentioned earlier is applicable only to weak-to-moderate planar shock waves and relatively high loading ratios, is shown in Fig. 2 for $M_0 = 1.5$, $D = 200 \mu\text{m}$, and $\eta = 4.0$. It is evident that the general attenuation law as developed in the course of the present study predicts an attenuation similar to that predicted by Olim et al.'s¹ limited law. The difference of about 3 to 4%, which is developed at large values of x , is due to the fact that whereas the present general attenuation law dictates a realistic equilibrium shock wave Mach number, $M_e \geq 1$, Olim et al.'s¹ specific attenuation law inherently dictates $M_e = 1$ when $x \rightarrow \infty$.

Conclusions

The governing equations describing the flowfield that develops when a planar shock wave propagates inside a dust-gas suspension were solved numerically using the random choice method.

The shock wave attenuation was investigated numerically. Based on the numerical results, a general law for describing the shock wave attenuation as it propagates inside the dust-gas suspension was developed. It is an improvement of a simplified specific attenuation law that was developed a few years ago by Olim et al.¹ The results indicated that the coefficient of attenuation χ decreases as the dust particles diameter D decreases and the dust loading ratio η increases. As a result the attenuation itself increases when D decreases and η increases.

The general attenuation law was validated by comparing its prediction with actual experimental results. Very good agreement was evident.

The general attenuation law enables one to calculate simply and cheaply the instantaneous shock wave Mach number and as a result to obtain immediately the jump conditions across it without the need to conduct a tedious numerical simulation.

Acknowledgment

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Interaction of a Regular Reflection with a Compressive Wedge: Analytical Solution

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Introduction

An experimental and analytical study of the reflection processes of planar shock waves over double wedges was presented in Ref. 1. One out of the seven investigated double wedge combinations is shown in Fig. 1. The slopes of the first and second reflecting surfaces are θ_w^1 and θ_w^2 , respectively, and θ_w^1 is large enough to ensure that the incident planar shock wave reflects over it as a regular reflection (RR). The case for which $\theta_w^2 < 90^\circ$ was investigated experimentally in Ref. 1 and numerically in Ref. 2, and the case

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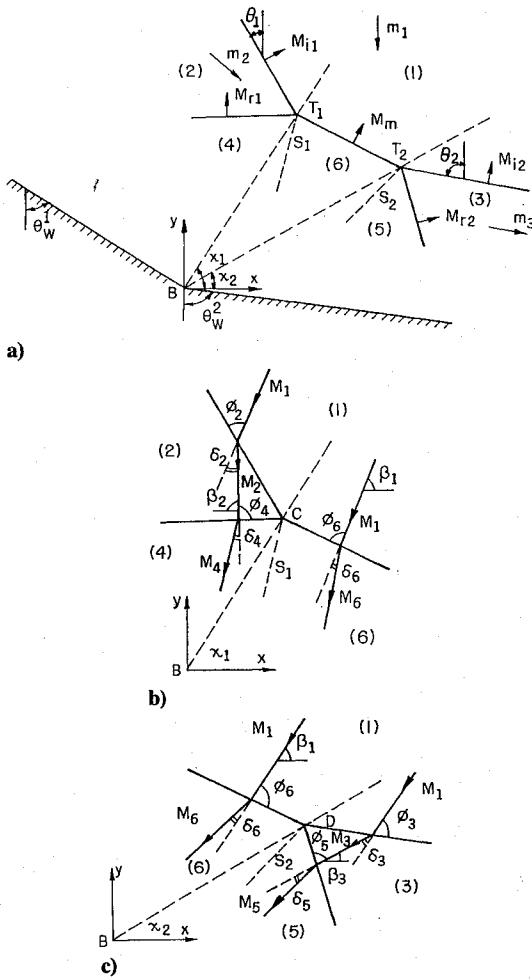


Fig. 3 Detailed schematic illustrations of the wave configurations shown in Fig. 2 with detailed definition of parameters.

Based on Fig. 3b and simple vector analysis one can write

$$M_1(C) = (M_{C1}^2 + m_1^2 + 2M_{C1}m_1 \sin \chi_1)^{\frac{1}{2}} \quad (4)$$

$$M_2(C) = [M_{C2}^2 + m_2^2 + 2M_{C2}m_2 \sin (\chi_1 - \theta_w^1)]^{\frac{1}{2}} \quad (5)$$

$$\beta_1(C) = \arctg \left(\frac{M_{C1} \sin \chi_1 + m_1}{M_{C1} \cos \chi_1} \right) \quad (6)$$

$$\beta_2(C) = \arctg \left(\frac{M_{C2} \sin \chi_1 + m_2 \cos \theta_w^1}{M_{C2} \cos \chi_1 - m_2 \sin \theta_w^1} \right) \quad (7)$$

From gas dynamic considerations one obtains

$$\delta_i(C) = \arctg \left\{ 2 \cotg \phi_i(C) \frac{[M_j(C) \sin \phi_i(C)]^2 - 1}{M_j^2(C) [\gamma + \cos 2\phi_i(C)] + 2} \right\} \quad (8) \text{ and } (9)$$

$$p_i(C) = p_j(C) \frac{2\gamma [M_j(C) \sin \phi_i(C)]^2 - (\gamma - 1)}{\gamma + 1} \quad (10) \text{ and } (11)$$

where $i = 4$ and 6 for $j = 2$ and 1 , respectively. In addition,

$$M_m = M_1(C) \sin \phi_6(C) \quad (12)$$

$$\theta_m = \beta_1(C) + \phi_6(C) - (\pi/2) \quad (13)$$

where M_m and θ_m are the strength and orientation of the Mach stem.

The matching conditions across the slipstream s_1 are

$$p_4(C) = p_6(C) \quad (14)$$

$$\beta_2(C) - \delta_4(C) = \beta_1(C) + \delta_6(C) \quad (15)$$

In addition, since the tangential velocity across oblique shock waves remains constant,

$$m_2 = \frac{m_1 \cos \theta_1}{\cos \omega_1} \frac{a_1}{a_2} \quad (16)$$

Equations similar to Eqs. (1–16) can be derived for the second triple point D .

Based on Fig. 3a the attachment velocity of the second triple point D is

$$v_D = \frac{M_{i2} - m_1 \sin \theta_2}{\cos(\chi_2 - \theta_2)} \cdot a_1 \quad (17)$$

where θ_2 is defined in Fig. 3a. Let us again define two different Mach numbers for this velocity

$$M_{Di} = v_D/a_i \quad i = 1, 3 \quad (18) \text{ and } (19)$$

Based on Fig. 3c and simple vector analysis one can write:

$$M_1(D) = (M_{D1}^2 + m_1^2 + 2M_{D1}m_1 \sin \chi_2)^{\frac{1}{2}} \quad (20)$$

$$M_3(D) = [M_{D3}^2 + m_3^2 + 2M_{D3}m_3 \sin(\chi_2 - \theta_w^2)]^{\frac{1}{2}} \quad (21)$$

$$\beta_1(D) = \arctg \left(\frac{M_{D1} \sin \chi_2 + m_1}{M_{D1} \cos \chi_2} \right) \quad (22)$$

and

$$\beta_3(D) = \arctg \left(\frac{M_{D3} \sin \chi_2 + m_3 \cos \theta_w^2}{M_{D3} \cos \chi_2 - m_3 \sin \theta_w^2} \right) \quad (23)$$

From gas dynamic considerations one obtains

$$\delta_i(D) = \arctg \left\{ 2 \cotg \phi_i(D) \frac{[M_j(D) \sin \phi_i(D)]^2 - 1}{M_j^2(D) [\gamma + \cos 2\phi_i(D)] + 2} \right\} \quad (24) \text{ and } (25)$$

$$p_i(D) = p_j(D) \frac{2\gamma [M_j(D) \sin \phi_i(D)]^2 - (\gamma - 1)}{\gamma + 1} \quad (26) \text{ and } (27)$$

where $i = 5$ and 6 for $j = 3$ and 1 , respectively. In addition,

$$M_m = M_1(D) \sin \phi_6(D) \quad (28)$$

$$\theta_m = \beta_1(D) - \phi_6(D) + (\pi/2) \quad (29)$$

The matching conditions across the slipstream s_2 are

$$p_5(D) = p_6(D) \quad (30)$$

$$\beta_3(D) + \delta_5(D) = \beta_1(D) - \delta_6(D) \quad (31)$$

In addition, since the tangential velocity across oblique shock waves remains constant,

$$m_3 = \frac{m_1 \cos \theta_2}{\cos \omega_2} \frac{a_1}{a_3} \quad (32)$$

Table 1 Comparison between analytical predictions and experimental results

	Case 1, $M_s = 1.25$, $\theta_w^1 = 55$ deg, $\theta_w^2 = 90$ deg		Case 2, $M_s = 1.49$, $\theta_w^1 = 55$ deg, $\theta_w^2 = 90$ deg	
	Analysis	Experiment	Analysis	Experiment
χ_1 , deg	57.6	52 deg ± 1	56.9	52 deg ± 1
χ_2 , deg	39.1	44 deg ± 1	34.7	37.5 deg ± 1
θ_m , deg	56.6	56 deg ± 1	58.4	58 deg ± 1

The set of 32 governing equations consists of the following 32 unknowns: v_C , M_{C1} , M_{C2} , $M_1(C)$, $M_2(C)$, $\beta_1(C)$, $\beta_2(C)$, $\delta_4(C)$, $\delta_6(C)$, $p_4(C)$, $p_6(C)$, M_m , θ_m , m_2 , $\phi_4(C)$, $\phi_6(C)$, χ_1 , v_D , M_{D1} , M_{D2} , $M_1(D)$, $M_3(D)$, $\beta_1(D)$, $\beta_3(D)$, $\delta_5(D)$, $\delta_6(D)$, $p_5(D)$, $p_6(D)$, m_3 , $\phi_5(D)$, $\phi_6(D)$ and χ_2 .

Note that the speed of sound a_1 behind the incident shock wave is simply obtained from M_s and a_0 . The parameters M_{i1} , θ_1 , a_2 , and ω_1 can be obtained from solving the regular reflection of incident shock wave M_s over the first reflecting surface θ_w^1 . Similarly, M_{i2} , θ_2 , a_3 , and ω_2 can be obtained from solving the regular reflection of the incident shock wave M_s over the second reflecting surface θ_w^2 . Recall that M_s , θ_w^1 , θ_w^2 , and the flow state (0) are all known, as they are the initial conditions.

Results and Discussion

Predictions of the proposed analytical model were compared with the relevant experimental results of Refs. 1–3. Three geometrical parameters were compared: the first and second triple point trajectory angles χ_1 and χ_2 and the orientation of the Mach stem θ_m with respect to the horizontal x axis. The comparison is shown in Table 1. Whereas the analytical predictions of χ_1 overestimate the experimental results by about 10%, the analytical predictions of χ_2 underestimate the experimental results by about 10%. Although these agreements do not seem to be too good, one should recall that similar agreement is obtained when the three-shock theory is used to predict the triple-point trajectory angle in Mach reflections over single wedges.⁴ The reason for this disagreement lies in the fact that in actual triple points not all of the shock waves are straight as required by the two- and three-shock theories. Figures 2a and 2b clearly indicate this fact. Furthermore, whereas in the case of single wedges the triple point moves toward a quiescent gas in the wave configuration treated in the present study, the triple points move toward a moving gas. Consequently, the present problem is much more complicated and, hence, an agreement within 10% should practically be considered as a very good one. Finally, it should be noted that as is evident from Table 1, the agreement between the presently proposed analytical model and the experimental results, regarding the orientation of the Mach stem, i.e., θ_m , is excellent.

Conclusions

The two- and three-shock theories were applied to complex flow-fields and wave structures. Their performance was found to be good to excellent. By further developing this model the pressures acting on the surfaces can be estimated.

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Effect of Screen Porosity and Location on Wind-Tunnel Turbulence

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Introduction

ONE of the fundamental problems of engineering fluid mechanics is how to control the velocity distribution of fluid flow inside the wind tunnel. Often, single screen or multiple screens are used in this operational mode to remove or create time-mean velocity nonuniformities and to reduce or increase the intensity of turbulence in a controlled manner. Numerous studies have investigated the effect of placing screens in the fluid flow through wind tunnels since the beginning of this century. Furthermore, the researchers have examined the effect of the turbulence intensity, as controlled by the screens, on forced convection heat transfer results.

A wide variety of turbulence generators have been examined in the past, such as square-mesh arrays of either round rods or wires (woven screens), square-mesh arrays of square bars, parallel arrays of square bars, perforated plates, agitated bar grids, jet grids, aerofoil cascades, tube bundles, and various permutations and combinations of the preceding. Roach¹ investigated the pressure drop across screens and the characteristics of the downstream turbulence. Furthermore, Roach¹ attempted to fill gaps in the current literature by proposing simple rules for the design of screens in wind tunnels. Therefore, he proposed design guidelines and also examined the pressure losses, turbulence intensities, spectra, correlation functions and length scales. In addition, Roach¹ introduced a number of correlations to predict turbulence intensity behind a screen; however, he did not address the effect of screen porosity on turbulence intensity. Laws and Livesey² investigated the flow through screens by characterizing the flow properties of the screen, by determining the effect of a screen on time-mean velocity distributions, and by measuring the turbulence distribution downstream of gauze screens. Furthermore, Gad-el-Hak and Corrsin³ gave details of turbulence intensities, scales, decays, and spectra for their jet grid. In the same paper they summarized the results, in tabular form, of no less than 12 previous paper on both passive and active grids giving the turbulence component magnitudes and decays of the form

$$\cot u/u \times 100Tu = B(x/M)^{-m} \quad (1)$$

where B and m are constants, x is the distance between the location of the screen and the measurement location of Tu , and M is square cell width of the screen based on wire centerline. Batchelor and Townsend⁴ and Compte-Bellot and Corrsin⁵ correlated their

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